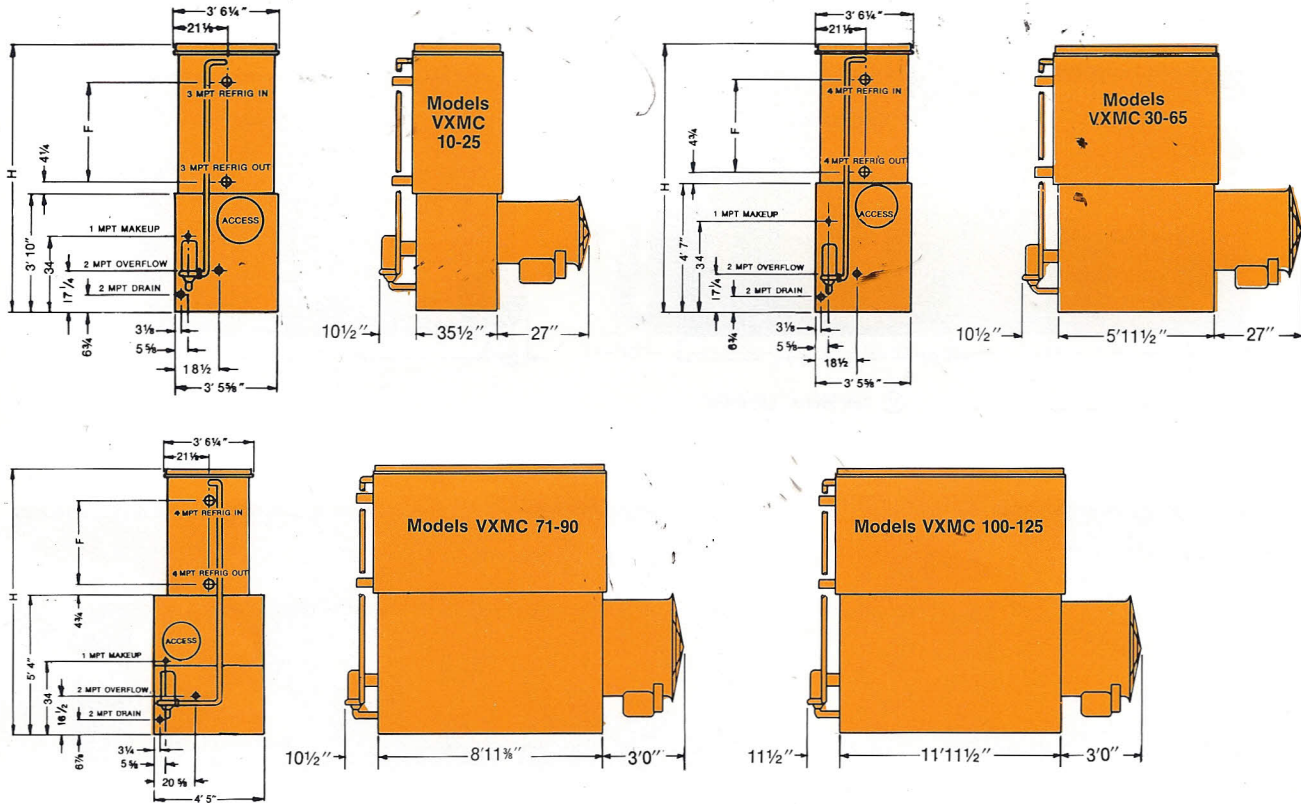


Engineering Data / Models VXMC 10

Do not use for construction. Refer to factory certified dimensions.

This brochure includes data current at the time of publication which should be reconfirmed at the time of purchase.



MODEL NO.	APPROX. SHPG. WEIGHT	APPROX. OPER. WEIGHT	HEAVIEST SECTION (COIL)	CFM	FAN MOTOR HP (0" ESP)	GPM	PUMP MOTOR HP	R-717 CHARGE (LBS.)	REMOTE SUMP		F	H
									BOTTOM DRAIN SIZE	APPROX. OPER. WEIGHT		
VXMC 10	1070	1380	1070*	2900	1/3	35	1/3	19	2 1/2	1140	15 3/4	80%
VXMC 15	1260	1580	850	3800	1/3	35	1/3	25	2 1/2	1340	25 1/4	90%
VXMC 20	1420	1760	1000	4400	1	35	1/3	32	2 1/2	1520	34 3/4	99%
VXMC 25	1470	1810	1050	5300	2	35	1/3	34	2 1/2	1570	34 3/4	99%
VXMC 30	1850	2550	1120	8200	2	75	1/2	35	3	2090	14 3/4	89%
VXMC 38	2100	2810	1350	8900	2	75	1/2	45	3	2350	24 1/4	99%
VXMC 46	2390	3130	1650	8500	2	75	1/2	61	3	2670	33 3/4	108%
VXMC 51	2440	3180	1700	10000	3	75	1/2	65	3	2720	33 3/4	108%
VXMC 57	2700	3480	1940	9600	3	75	1/2	76	3	3020	43 1/4	118%
VXMC 65	2770	3550	2010	11600	5	75	1/2	80	3	3090	43 1/4	118%
VXMC 71	3670	4980	2400	12100	3	115	3/4	90	4	4230	37 3/4	121 7/8
VXMC 80	3740	5050	2470	14500	5	115	3/4	100	4	4300	37 3/4	121 7/8
VXMC 90	4160	5270	2850	14000	5	115	3/4	110	4	4520	48 1/4	132 3/8
VXMC 100	4600	6370	3060	19600	5	150	1	120	6	5360	37 3/4	121 7/8
VXMC 110	4680	6450	3140	22000	7 1/2	150	1	130	6	5440	37 3/4	121 7/8
VXMC 125	5230	7060	3640	21000	7 1/2	150	1	145	6	6050	48 1/4	132 3/8
VXMC 138	7130	8920	4920	25900	5	220	1 1/2	170	6	7510	37 3/4	133 7/8
VXMC 150	7220	9010	5830	28300	7 1/2	220	1 1/2	190	6	7590	37 3/4	133 7/8
VXMC 170	8120	9930	5930	28000	7 1/2	220	1 1/2	210	6	8510	48 1/4	144 3/8
VXMC N195	9950	13540	6580	40000	7 1/2	305	3	240	6	11100	37 3/4	145 7/8
VXMC N215	11190	14840	7810	37900	7 1/2	305	3	295	6	12400	48 1/4	156 3/8
VXMC N235	11330	14970	7950	42300	10	305	3	320	6	12530	48 1/4	156 3/8
VXMC N265	15230	20790	10180	54800	7 1/2	460	5	360	8	17070	37 3/4	157 7/8
VXMC N285	15360	20920	10320	58600	10	460	5	390	8	17200	37 3/4	157 7/8
VXMC N315	17160	22790	12060	56000	10	460	5	440	8	19070	48 1/4	168 3/8
VXMC N345	17370	22980	12270	61700	15	460	5	470	8	19270	48 1/4	168 3/8

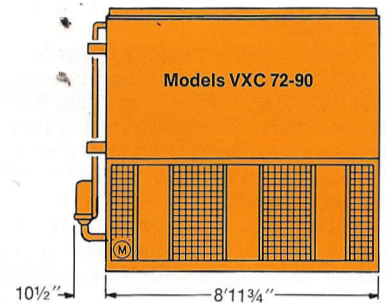
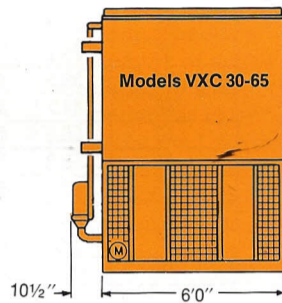
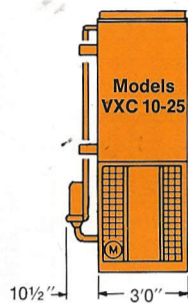
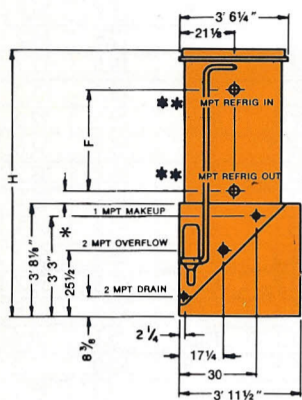
* Unit normally ships in one piece.

Engineering Data / Models VXC 10

Do not use for construction. Refer to factory certified dimensions.

This brochure includes data current at the time of publication which should be reconfirmed at the time of purchase.

Models VXC 10-135

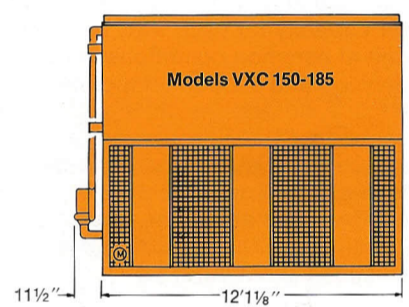
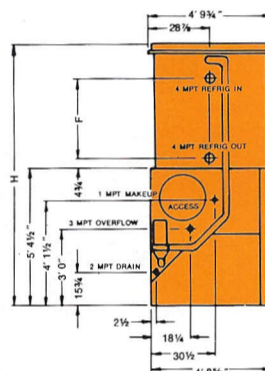
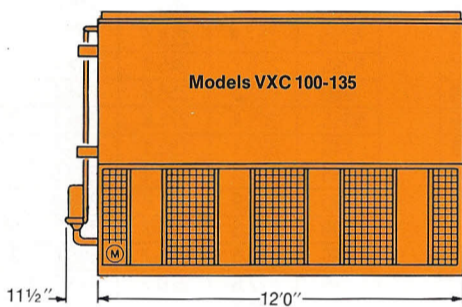


(M) Fan Motor Location

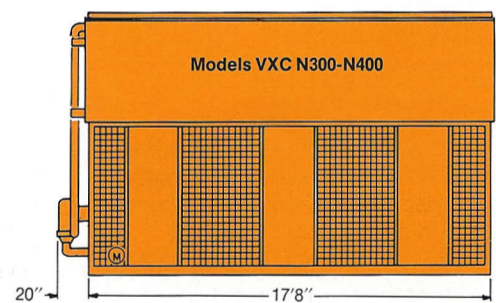
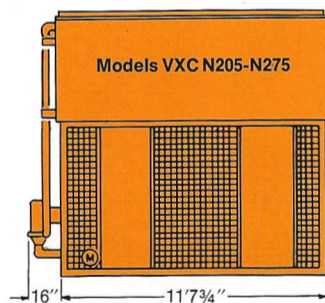
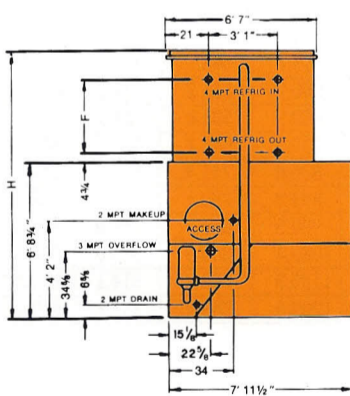
* 4 1/4" on VXC 10 through 25
4 3/4" on VXC 30 and larger

** See Note 2

Models VXC 150-185 only



Models VXC N205-N400



Selection

Two methods of selection are presented in this section, the *heat rejection method* shown on these two pages, and the *evaporator ton method* shown on Pages 14 and 15. Selections may be made from the heat rejection method for any type of positive displacement compressor: open reciprocating, hermetic reciprocating, or rotary screw. The evaporator ton method is based on evaporator heat input only, and is limited to systems utilizing open reciprocating compressors.

Heat Rejection Method

In a mechanical refrigeration system, the function of an evaporative condenser is to reject heat to the environment. The heat to be rejected is the sum of the heat input at the evaporator and the energy input at the compressor. For a given set of operating conditions, the energy input through the compression process can vary for the several types of compressors—centrifugal, rotary screw, open reciprocating, and hermetic reciprocating. Therefore, in order to accurately determine the proper evaporative condenser required, it is necessary to establish the compressor energy input as well as the heat absorbed in the evaporator.

Frequently the total heat rejection of a system is specified. When it is not specified, it can be readily calculated. Total heat rejection is the sum of the compressor evaporator capacity in BTUH at the specified operating conditions, and the energy corresponding to the compressor brake horsepower in BTUH.

For open compressors:

$$\text{Total heat rejection} = \text{Compressor evaporator capacity (BTUH)} + \text{Compressor BHP} \times 2545$$

TABLE 1 – Base Heat Rejection – Model VXC

(MBH — THOUSANDS OF BTU'S PER HOUR)

MODEL NO. VXC	HEAT REJECTION MBH	MODEL NO. VXC	HEAT REJECTION MBH	MODEL NO. VXC	HEAT REJECTION MBH
10	147.0	185	2,719.5	590	8,673.0
15	220.5	N205	3,013.5	N600	8,820.0
20	294.0	N230	3,381.0	620	9,114.0
25	367.5	N250	3,675.0	650/N650	9,555.0
30	441.0	N275	4,042.5	680	9,996.0
38	558.6	N300	4,410.0	720/N720	10,584.0
46	676.2	320	4,704.0	760/N760	11,172.0
52	764.4	N325	4,777.5	N800	11,760.0
58	852.6	340	4,998.0	840	12,348.0
65	955.5	360/N360	5,292.0	900	13,230.0
72	1,058.4	380/N380	5,586.0	980	14,406.0
80	1,176.0	N400	5,880.0	1060	15,582.0
90	1,323.0	420	6,174.0	1100	16,170.0
100	1,470.0	450	6,615.0	1180	17,346.0
110	1,617.0	N460	6,762.0	1240	18,228.0
125	1,837.5	490	7,203.0	1300	19,110.0
135	1,984.5	N500	7,350.0	1360	20,000.0
150	2,205.0	530	7,791.0		
165	2,425.5	550/N550	8,085.0		

For multi-stage open compressor systems, total heat rejection is calculated from the *high stage* compressor capacity and brake horsepower, expressed in BTUH.

In the case of hermetic compressors, compressor input is commonly expressed in KW and must be converted to BTUH:

$$\text{Total heat rejection} = \text{Compressor evaporator capacity (BTUH)} + \text{Compressor KW} \times 3415$$

The base heat rejection of each Baltimore Aircoil evaporative condenser is shown in Tables 1 and 2. This represents the total heat rejection of each unit when operating at 105°F condensing temperature and 78°F wet bulb temperature, using refrigerants R-12, R-22, R-500, or R-502. Tables 3 and 4 present correction factors to be applied to the system heat rejection for other operating conditions of condensing temperature, wet bulb temperature, and refrigerant.

VXC and VXMC units which have the letter "N" preceding the model number have a maximum width of eight (8) feet at the base. Units which do not have the letter "N", and have model numbers greater than 185, are ten (10) feet wide at the base.

Selection Procedure

1. Establish total heat rejection required by the system (See above).
2. Determine the refrigerant and design conditions for condensing temperature and wet bulb temperature.
3. Using the appropriate factor (Tables 3 and 4) for the proper refrigerant, determine the correction factor to be applied to the system heat rejection.
4. Multiply the correction factor by the total system heat rejection.

TABLE 2 – Base Heat Rejection – Model VXMC

(MBH — THOUSANDS OF BTU'S PER HOUR)

MODEL NO. VXMC	HEAT REJECTION MBH	MODEL NO. VXMC	HEAT REJECTION MBH	MODEL NO. VXMC	HEAT REJECTION MBH
10	147.0	150	2,205.0	N530	7,791.0
15	220.5	170	2,499.0	560	8,232.0
20	294.0	N195	2,866.5	N570	8,379.0
25	367.5	N215	3,160.5	585	8,599.5
30	441.0	N235	3,454.5	600	8,820.0
38	558.6	N265	3,895.5	620	9,114.0
46	676.2	N285	4,189.5	N630	9,261.0
51	749.7	300	4,410.0	680	9,996.0
57	837.9	N315	4,630.5	N690	10,143.0
65	955.5	340	4,998.0	760	11,172.0
71	1,043.7	N345	5,071.5	860	12,642.0
80	1,176.0	380	5,586.0	920	13,524.0
90	1,323.0	N390	5,733.0	1020	14,994.0
100	1,470.0	430/N430	6,321.0	1120	16,464.0
110	1,617.0	460	6,762.0	1170	17,199.0
125	1,837.5	N470	6,909.0	1240	18,228.0
138	2,028.6	510	7,497.0		